GAUTENG DEPARTMENT OF EDUCATION

SENIOR CERTIFICATE EXAMINATION

TECHNIKA (MECHANICAL) SG

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OCTOBER / NOVEMBER 2005 OKTOBER / NOVEMBER 2005

TIME: 3 hours

MARKS: 200

REQUIREMENTS:

- Pocket calculator
- Drawing instruments
- Information sheet (pages 9 to 12).

INSTRUCTIONS:

• ALL questions are COMPULSORY.

QUESTION 1A MULTIPLE-CHOICE QUESTIONS

Each of the following questions is supplied with various possible answers. Choose the correct answer or answers. Write down the question numbers below one another and ONLY the letter(s) of your choice next to them. The mark allocation at the right serves as an indication of the number of correct questions.

- 1.1 According to the law concerning factories, the worker's responsibility towards workshop safety is to ______.
 - A. perform all duties with the necessary precision
 - B. learn and apply safe working methods
 - C. obey all rules and regulations
 - D. do exactly what he/she is told to do

1.2 Screw-threads are used to _____.

- A. exert pressure
- B. permanently fasten plates
- C. make very minute adjustments
- D. None of the above

(2)

(2)

3

1.3	The purpose of intermediate gears is to								
	A. B. C. D.	increase the speed decrease the speed change direction of rotation make up space between fixed gears	(2)						
1.4	Friction are ex	n can be both advantageous or disadvantageous. Which of the following amples of the advantageous use of friction in a workshop?							
	A. B. C. D. E.	Cutting tools Belt drives Gear drives Brakes on machines Clutches on machines	(3)						
1.5	The ex for	tension spindle is an important component of a dividing head. It is used							
	A. B. C. D.	the set-up to cut hexagons the mounting of change gears for helical milling the extension of the spindle to accommodate more than one gear angular indexing	(1)						
1.6 Single helical gear drives have advantages and disadvantages. Which of the following are advantages?									
	A. B. C. D.	Power transmission is distributed over at least two teeth. It can cause end thrust. The drive is fairly quiet. It is frictionless.	(2)						
1.7	Which speed	of the following should be taken into account when determining the of the table feed of a milling machine?							
	A. B. C. D.	Diameter of cutter Dimensions of the work Condition of the machine The power of the machine	(2)						
1.8	Accide Which	ents in a workshop are caused by either personal or work-related factors. of the following are personal factors?							
	A. B. C. D.	No knowledge/skills Broken-down machines Wrong attitudes Faulty electrical wiring	(2)						

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	are	·						
	A. B. C. D.	the lead of the thread the primary clearance angle of the cutting tool the root diameter of the screw-thread the outside diameter of the workpiece	(2)					
1.10	The fu	nction of a milling machine tailstock is that						
	A. B. C. D.	it supports the workpiece whilst cutting gears it clamps the workpiece whilst cutting gears it supports the workpiece whilst cutting small tapers None of the above	(1)					
1.11	Which	of the following methods are used to test tapers on shafts?						
	A. B. C. D.	Between lathe centres V-blocks and clamps Squares and rulers Height gauges and surface plates	(2)					
1.12	.12 The aim of maintenance in a workshop is to							
	A. B. C. D.	cut the cost of replacement prevent workers from working overtime improve the quality/design of equipment keep workers on their toes	(2)					
1.13	Unbala the eff	anced workpieces are detrimental to lathes. Which of the following are ects of unbalanced workpieces?						
	A. B. C. D.	Damaged workpieces Damaged lathe beds Damaged spindle bearings Distorted face plates	(2)					
1.14	The ac	dvantages of multi-start threads over single-start threads are that						
	A. B. C. D.	smaller pitches can be cut movements are quicker shallower threads and thus stronger thread spindles are acquired larger diameter workpieces can be used	(2)					

The factors that influence the magnitude of the helix angle of a screw-thread

1.9

1.15	Helical gears can transmit more power than spur gears. The reason for this is that										
	A. B. C. D.	the helix angle on helical gears is larger helical gears can be cut on much larger diameter shafts power transfer is distributed over two teeth the drive is silent	(1)								
1.16	Comp	pressive stress is applicable when									
	A. B. C. D.	two objects are pressed together an object is being stretched an object is held in a bench vice two surfaces slide over each other	(2)								
1.17	Choo	se the disadvantageous uses of friction from the list below.									
	A. B. C. D.	Belt drives Chain drives Brakes on machines Drill bits	(2)								
1.18	The a	dvantages of climb milling are that									
	A. B. C. D.	the work is forced against the vice or table less energy is used cutters stay sharp for a longer period it helps to eliminate backlash	(2)								
1.19	The b	acklash on gears is the									
	A. B. C. D.	difference between the modules of the meshing gears cause of chattering on gears play between two meshing gears cause of gears not meshing properly	(1)								
1.20	The fr Identi	ictional resistance between two sliding surfaces depends on three factors. fy these factors.									
	A. B. C. D.	The types of materials of the two sliding surfaces The magnitude of the frictional force between the surfaces The degree of coarseness of the two surfaces The magnitude of the contact surfaces	(3)								

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1.21	Which one of the following metals is NOT part of the cubic space-lattice arrangement?								
	A. B. C. D. E.	Aluminium Copper Lead Magnesium Platinum	(1)						
1.22	What is with th	s the correct name of the hydrocarbon compound in the alkane series e formula C_8H_{18} ?							
	A. B. C. D. E.	Heptane Heksane Octane Propane Butane	(1) [40]						
		QUESTION 2							
2.1	Explair	n how AIDS can be spread from one person to another.	(4)						
2.2	Name Aids cr	TWO ways in which you can make your employees aware of the growing isis in our country.	(2)						
2.3	A load drum is	of 1,7 kN is lifted by the drum of a lifting device. The diameter of the s 0,2 m. Calculate the power if the drum rotates at 16 r/min.	(4)						
2.4	A mild The sti	steel tie-bar is 4,5 m long and is subjected to a tensile load of 600 kN. ress in the bar is 570 MPa. Determine the diameter of the bar in mm.	(10)						
2.5	Define	the following concepts:							
	2.5.1 2.5.2 2.5.3 2.5.4 2.5.5 2.5.6 2.5.7	Mass Weight Gravitational force Hooke's Law Young's Elasticity Modulus Boyle's Law Thermodynamics	(2) (3) (4) (4) (3) (2) [40]						

QUESTION 3

3.1	The fluid pressure in a cylinder is 6 MPa and the diameter of the cylinder is 0,54 m. Calculate the power exerted when the piston moves outwards.											
3.2	Define the following concepts:											
	 3.2.1 Pressure 3.2.2 Pascal's Law 3.2.3 Hydraulics 3.2.4 Viscosity of fluids 	(3) (4) (4) (2)										
3.3	Name FIVE properties of fluids.											
3.4	Why must a dam-wall be thicker at the bottom than at the top?	(1)										
3.5	Name SIX occupational diseases and name the cause of each.	(12)										
3.6	What is the main aim of the Factory Act?											
	QUESTION 4											
4.1	Why is differential indexing necessary?											
4.2	Calculate the indexing for each of the following cases:											
	4.2.1 A gear with 132 teeth	(3)										
	4.2.2 Differential indexing for 51 divisions											
	 a) Calculate the indexing required. (Choose 50 divisions) b) Calculate the change wheels required. c) Determine the direction of rotation of the index plate. d) Draw a simple sketch to clearly show the position and arrangement of the change gears. 	(3) (5) (1) (3)										
4.3	Give the name of the milling cutter suitable for milling a wide flat surface.	(1)										
4.4	Define the concept ergonomics .	(4)										
4.5	Name THREE basic crystal structures of steel.											
4.6	Name FIVE types of keys in common use.											
4.7	Calculate the width, length and thickness of an ordinary parallel key that is used to secure a component to a 30 mm diameter shaft.	(6)										
4.8	List FOUR characteristics of a good business leader.											

QUESTION 5

	TOTAL:	200						
		[40]						
5.8	Describe how a direct-drive gear ratio can be obtained by means of a single epicyclic gear train.							
5.7	Draw a simple schematic diagram of a single epicyclic gear train.							
5.6	Draw a simple sketch of a Prony brake.	(8)						
5.5	Define power .	(2)						
	Calculate the indicated power of the engine in kW.	(7)						
	 Mean effective pressure - 850 KPa Stroke 70 mm Cylinder bore 80 mm Crankshaft revolutions 3 600 per minute Number of cylinders 4 							
5.4	The following information concerns a four-stroke engine:							
5.3	An engine develops a torque of 200 Nm at 3 000 revolutions per minute. Calculate the brake power in kW.							
5.2	Define brake power .							
5.1	What is the difference between velocity and speed ?	(4)						

INFORMATION PAGES / INLIGTINGSBLADSYE

1. Tooth gears for milling machine / Tandratte vir freesmasjien Standard and special wheels / Standaard- en spesiale wiele

24 (two of these / twee van hierdie); 28; 32; 40; 44; 46; 47; 48; 52; 56; 58; 64; 68; 70; 72; 76; 84; 86 and/en 100 teeth / tande

2. Index plate for milling machine / Indeksplaat vir freesmasjien

Standard Cincinnati index machine / Standaard-Cincinnati-indeksmasjien 24; 25; 28; 30; 34; 37; 38; 39; 41; 42; 43; 46; 47; 49; 51; 53; 54; 57; 58; 59; 62 and/en 66 holes/gate

- 3. Take p = 3,14 / Neem p = 3,14
- 4. Take $g = 10 \text{ m.s}^{-2} / \text{Neem } g = 10 \text{ m.s}^{-2}$
- 5. Formulae / Formules
 - 5.1 Indexing / Indeksering:
 - 5.1.1 Simple indexing / Eenvoudige indeksering = $\frac{40}{N}$

[Dr = Drive gear / Dryfrat] [Dn / Gd = Driven gear / Gedrewe rat]

5.1.2 Differential indexing / Differensiaalindeksering

$$= \frac{\mathbf{Dr}}{\mathbf{Gdr}} = \frac{(\mathbf{A} - \mathbf{N})}{\mathbf{A}} \times \frac{40}{1}$$

5.2 Two-wire method of screw-thread measurement / Tweedraadmetode van skroefdraadmeting:

Calculation of included angle / Berekening van ingeslote hoek:

$$\sin \frac{?}{2} = \frac{\mathbf{R} - \mathbf{r}}{(\mathbf{M} - \mathbf{m})} + \mathbf{r} - \mathbf{R}$$

5.3 Friction: Co-efficient of friction / Wrywing: Wrywingskoëffisiënt $\mu = \frac{F\mu}{R}$

5.4 Stress / Spanning = $\frac{F}{A}$

- 5.5 Cross-sectional area of solid cylinder / Dwarsdeursnee-area van soliede silinder = pD^2 or/of πr^2
- 5.6 Cross-sectional area of hollow cylinder / Dwarsdeursnee-area van hol silinder = $\frac{p(D^2 d^2)}{4}$

- 5.8 Strain = Change in length Original length /Vormverandering = Verandering in lengte Oorspronklike lengte
- 5.9 Factor of Safety = Ultimate stress /Veiligheidsfaktor = Breekspanning Working stress

5.10 Angular acceleration / Hoekversnelling = $\frac{2}{t} = \frac{2}{t}$

- 5.11 Torque T / Draaimoment T = mk²?²
- 5.12 Moment of inertia / Traagheidsmoment $I = mk^2$
- 5.13 Angular velocity / Hoeksnelheid $? = \frac{2\pi N}{60}$
- 5.14 Kinetic energy of a flywheel / Kinetiese energie van ? vliegwiel $E_k = \frac{1}{2} \text{ mk}^2$?²
- 5.15 Belt drives / Bandaandrywings

5.15.1 Power P / Drywing P – $(T_1 – T_2) \pi Dn$

5.15.2 $D_{Dr} \times N_{Dr} = D_{DN} \times N_{DN}$ (Dr = Driver pulley) (Dn = Driven pulley)

 $D_{Dr} \times N_{Dr} = D_{Gdr} \times N_{Gdr} (Dr = Dryfkatrol)$ (Gdr = Gedrewe katrol)

5.16 Gear drives / Rataandrywings

5.16.1 $N_A \times T_A = N_B \times T_B$

715-2/0 L

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5.16.2 Revolution s of final driven gear / Omwentelin ge van finale gedrewe rat
Revolution s of first drive gear / Omwentelin ge van eerste dryfrat
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Product of number of teeth on all drive gears / Produk van getal tande op al die dryfratte Product of number of teeth on the driven gears Produk van getal tande op die gedrewe ratte

5.16.3 Speed ratio = Product of number of teeth on all drive gears Product of number of teeth on all driven gears

Spoedverhouding = Produk van getal tande van alle dryfratte Produk van getal tande van alle gedrewe ratte

5.17 Power / Drywing

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- 5.17.1 Indicated power IP = PLANn (N = Number of power strokes per second) Aangeduide drywing AD = PLANn (N = Getal kragslae per sekonde)
- 5.17.2 Brake power BP / Remdrywing RD = $\frac{2\pi NT}{60}$ 5.17.3 Torque T / Draaimoment T = Fr

5.17.4 Mechanical efficiency =
$$\frac{BP}{IP} \times \frac{100}{1} / Meganiese rendement \frac{RD}{AD} \times \frac{100}{1}$$

5.18 Motion equations / Bewegingsvergelykings

v = u + at	v = at	v = u + gt	v = gt
$s = ut + \frac{1}{2} at^2$	$s = \frac{1}{2} at^2$	$s = ut + \frac{1}{2}gt^2$	$s = \frac{1}{2}gt^2$
$v^2 = u^2 + 2as$	v ² = 2as	$v^2 = u^2 + 2gs$	v ² = 2gs

TECHNIKA (MECHANICAL) SG TECHNIKA (MEGANIES) SG 715-2/0 K	12

Nominal sizes Nominale groottes		CLEARANCE FITS VRY PASSINGS												TRANSITION FITS OORGANGPASSINGS				INTERFERENCE FITS SLUITPASSINGS			
		Tolerance Toleransie		erance Tolerance eransie Toleransie		ce Tolera sie Tolera		erance Tole eransie Tole		rance Tole ransie Tole		Toler <i>Toler</i>	Tolerance <i>Toleransie</i>		Tolerance <i>Toleransie</i>		rance <i>ansi</i> e	Tolerance Toleransie		Tolerance Toleransie	
Over Oor mm	To <i>Tot</i> mm	H11	c11	Н9	d10	H9	e9	H8	F7	H7	g6	H7	h6	H7	k6	H7	n6	H7	р6	H7	s6
										UNIT/	EENHE	EID 0,00	1 mm								
10	18	+ 110	- 95	+ 43	- 50	+ 43	- 32	+ 27	- 16	+ 18	- 6	+ 18	- 11	+18	+ 12	+18	+ 23	+ 18	+ 29	+ 18	+ 39
		0	- 205	0	- 120	0	- 75	0	- 34	0	- 17	0	0	0	+ 1	0	+ 12	0	+ 18	0	+ 28
18	30	+ 130	- 110	+ 52	- 65	+ 52	- 40	+ 33	- 20	+ 21	- 7	+ 21	- 13	+ 21	+ 15	+ 21	+ 28	+ 21	+ 35	+ 21	+ 48
		0	- 204	0	- 149	0	- 92	0	- 41	0	- 20	0	0	0	+ 2	0	+ 15	0	+ 22	0	+ 35
30	40	+ 160	- 120																		
		0	- 280	+ 62	- 80	+ 62	- 50	+ 39	- 25	+ 25	- 9	+ 25	- 16	+ 25	+ 18	+ 25	+ 33	+ 25	+ 42	+ 25	+ 59
40	50	+ 160	- 130	0	- 180	0	- 112	0	- 50	0	- 25	0	0	0	+ 2	0	+ 17	0	+ 26	0	+ 43
		0	- 290																		

6. Table of primary fits (hole basis system) / Tabel van primêre passings (gatbasisstelsel)

Selection of Primary Fits (hole basis system) Seleksie van Primêre Passsings (gatbasisstelsel)